

IN THE SPECIFICATION:

Please replace paragraph [0002] with the following paragraph. Support for the additional description in this replacement paragraph can be found in the originally filed drawings and claims, and therefore no new matter is added by these changes.

In order to cool a rotor or remove its electrical heat losses in a generator, a (mostly gaseous) cooling medium flows through the rotor (see, for example, EP-A1-0 854 559). An important factor for good cooling is hereby the inflow geometry at the front faces of the rotor. A known inflow geometry is shown in Fig. 1A and 1B, whereby Fig. 1B shows an unwound view of the rotor front face, and Fig. 1A the section through the (not unwound) rotor front face in the plane A-A of Fig. 1B. The rotor 10 of a generator (13) comprises a central, cylindrical rotor body 18, around which one or more rotor winding(s) 14 is/are provided on the outside. The rotor winding 14 passes through the rotor active part 11 in several conductor bundles parallel to the rotor axis and makes a turn-around of 180° in the rotor winding head 12 between two conductor bundles. The widening space is closed off on the front face by an annular cap plate 19 (22 in Fig. 2 of EP-A1-0 854 559). End spacer plates 16 with the shape of a ring segment are provided between the cap plate 19 and the rotor winding 14. Cold cooling gas for cooling the rotor 10 flows into ring gap segments 33, 34 between the cap plate 19 and the rotor body 18. The ring gap segments 33, 34 are bordered, when seen in circumferential direction, on sides in each case by the section 16' of the end spacer plate 16 provided between the cap plate 19 and the rotor winding 14, whereby this section 16' is projecting into the ring gap. The section of the end spacer plates 16, whereby said section is projecting into the ring gap between the rotor winding 14 and rotor body 18, forms a continuous dividing wall together with two each axial chamber walls 15, 15', separating a cold gas chamber 25 and a warm gas chamber 32 with a gas inlet 21 and a gas outlet 22 (via corresponding ventilation grooves 20) from each other. The end spacer plate 16 is hereby used as a holder for the chamber walls 15, 15'. In most cases, this results in an overhang 24 that reaches into the cold gas chamber 25. In unfavorable cases, this overhang 24 even has an extension 23.

Please replace paragraph [0004] with the following paragraph to correct a minor typographical error:

In order to avoid such separations, the initially mentioned EP-A1-0 854 559 already suggested to provide a ~~glass~~ ~~gas~~ guide ring (GGR) in the form of a two-stage flow grate in the inlet area of the cold gas between the cap plate 19 and the rotor body 18. The GGR principally solves the ventilation technology problem, but has several disadvantages: on the one hand, the additional installation of such a GGR is associated with significant expenditure. On the other hand, its function is threatened as soon as significant changes are made in the cooling gas stream (e.g., for a shorter generator or lower performance). Finally, the function of the GGR depends on the rotation direction, so that the desired effect does not exist when the rotation direction is reversed.

CLAIM SUMMARY DOCUMENT:

1. (Currently amended) ~~{Turbo}~~ **A turbo** generator ~~[with]~~ **comprising** a rotor with direct gas cooling, ~~[which]~~ **said** rotor ~~[is provided with]~~ **including** a rotor winding arranged around a central rotor body, said rotor winding being **covered** on ~~[the]~~ front sides ~~[covered by one each]~~ **by an** annular cap plate, and in which rotor cold cooling gas for cooling the rotor flows into ring gap segments **of a ring gap** between the ~~[eap plate]~~ **rotor winding** and the rotor body, ~~[whereby]~~ **wherein** the ring gap segments are bordered~~[,~~ ~~when seen]~~ in **a circumferential direction**, on ~~[the sides]~~ **sides of the ring gap segments** in each case by ~~[the]~~ **a** section of an end spacer plate provided between the **annular** cap plate and the rotor winding, ~~[whereby this seetion is projecting]~~ **wherein said sections of said end spacer plates project** into the ring gap~~[,~~ ~~wherein]~~ **and are shaped to avoid** separations of the cooling gas stream ~~[on]~~ flowing into the ring gap ~~[segments are avoided by designing the sections of the end spacer plates, whereby these sections adjoin the ring gap, in a manner that is advantageous with respect to the flow]~~ **past said sections.**
2. (Currently amended) ~~{Turbo}~~ **The turbo** generator as claimed in Claim 1, ~~wherein[,~~ ~~when seen in circumferential direction, the]~~ side edges of the sections of the end spacer plates~~[,~~ ~~whereby said sections are]~~ **as viewed in a circumferential direction and** projecting into the cooling gas stream, are provided with either a bevel or bezel.
3. (Currently amended) ~~{Turbo}~~ **The turbo** generator as claimed in Claim 1, ~~wherein[,~~ ~~when seen in circumferential direction, the]~~ side edges of the sections of the end spacer plates~~[,~~ ~~whereby said seetions are]~~ **as viewed in a circumferential direction and** projecting into the cooling gas stream, are provided with a curvature with one or more curvature radii adapted to the **cooling gas** stream.
4. (Currently amended) ~~{Turbo}~~ **The turbo** generator as claimed in Claim 2, ~~[whereni]~~ **wherein** the end spacer plates with the beveled or rounded sections on the front

side close off a warm gas chamber defined on the sides by two axial, parallel chamber walls, and [that] the beveled or rounded sections terminate flush with the chamber walls.

5. (Currently amended) [Turbo] The turbo generator as claimed in Claim 2, wherein [one each additional] a gas guidance segment that guides the cooling gas stream around [the section] one of said sections of said end spacer plates is provided in the flow direction upstream from [the] each of said sections of [the] said end spacer plates.

6. (Currently amended) [Turbo] The turbo generator as claimed in Claim 5, wherein [the gas guidance segment is provided upstream from the section, separated from it] each of said gas guidance segments is separated in the upstream direction from one of said sections by a gap, and [that it] is provided with a convex curved outside facing the cooling gas stream.

7. (Currently amended) [Turbo] The turbo generator as claimed in Claim 6, wherein [the] each of said gas guidance [segment] segments has essentially the same length in a circumferential direction as [the section of the end spacer plate and forms a unit with the section with respect to flow technology.] said one of said sections of the end spacer plates and cooperates with the section in affecting the flow of the cooling gas stream.

8. (Currently amended) [Turbo] The turbo generator as claimed in Claim 6, wherein the section of the end spacer plate and the [preceding] upstream gas guidance segment are [designed] shaped so that cooling gas is pressed from the inlet side into the gap against the rotation direction of the rotor.

9. (Currently amended) [Turbo] The turbo generator as claimed in Claim 8, wherein the gas guidance segment in the rotation direction of the rotor projects with at least its front edge beyond the section of the end spacer plate.

10. (Currently amended) {Turbo} The turbo generator as claimed in Claim 8, wherein the gas guidance segment is {designed} shaped at its {edge, that is, in the rear-} {rear edge} in the rotation direction of the rotor, in such a way that the cooling gas stream flowing through the gap is added to the cooling gas stream flowing into {the} an adjoining ring gap segment essentially without a transition.

11. (Currently amended) {Turbo} The turbo generator as claimed in Claim 10, wherein the gas guidance segment is provided with an inwardly curved nose at the rear edge {located in the rear} in the rotation direction of the rotor.

12. (Currently amended) {Turbo} The turbo generator as claimed in Claim 5, wherein the gas guidance segment is attached to the annular cap plate.

13. (Currently amended) {Turbo} The turbo generator as claimed in Claim 2, wherein at least one of the {seetion} sections of the end spacer {plate} plates projects into the ring gap, has a greater thickness than {the} a remaining part of the end spacer plate and projects with its side facing the cooling gas stream into {the} a space {below} on a side of the annular cap plate facing the rotor winding.